

**FLEXIBLE AIRCRAFT DYNAMIC MODELING
FOR DYNAMIC ANALYSIS AND CONTROL SYNTHESIS**

By

David K. Schmidt
Purdue University
West Lafayette, Indiana

ABSTRACT

The linearization and simplification of a nonlinear, literal model for flexible aircraft is highlighted. Areas of model fidelity that are critical if the model is to be used for control system synthesis are developed and several simplification techniques that can deliver the necessary model fidelity are discussed. These techniques include both numerical and analytical approaches. An analytical approach, based on first-order sensitivity theory is shown to lead not only to excellent numerical results, but also to closed-form analytical expressions for key system dynamic properties such as the pole/zero factors of the vehicle transfer-function matrix. The analytical results are expressed in terms of vehicle mass properties, vibrational characteristics, and rigid-body and aeroelastic stability derivatives, thus leading to the underlying causes for critical dynamic characteristics.

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**MODELING
FLEXIBLE AIRCRAFT
FOR
DYNAMIC ANALYSIS
AND
CONTROL SYNTHESIS**

**David K. Schmidt
School of Aeronautics and Astronautics
Purdue University**

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TOPICAL OUTLINE

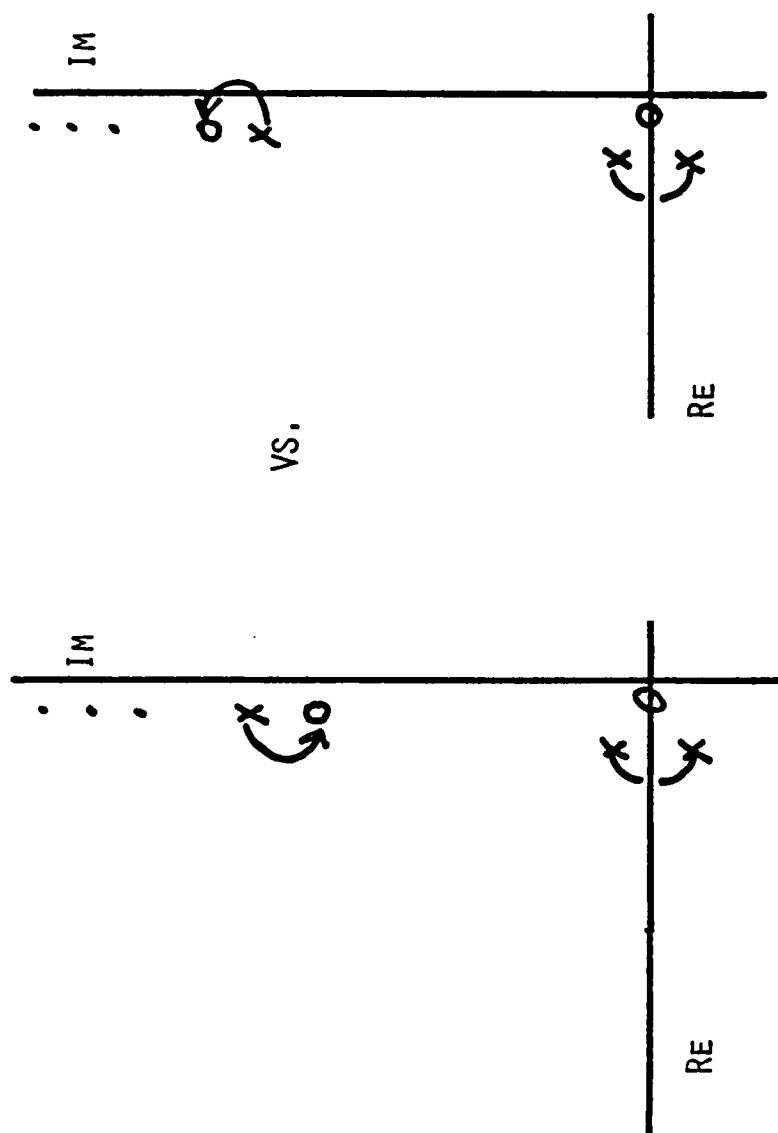
- **What Constitutes a Valid Model?**
 - How Validity Will Be Measured**
 - What's Important in a Feedback System**
- **Some Approaches to Obtain (Simple) Valid Models**
 - Numerical**
 - Literal**
- **Physical Causes of Critical Dynamic Characteristics**

WHAT VEHICLE DYNAMIC CHARACTERISTICS ARE CRITICAL IN A FEEDBACK SYSTEM?

They Are Important If They Can:

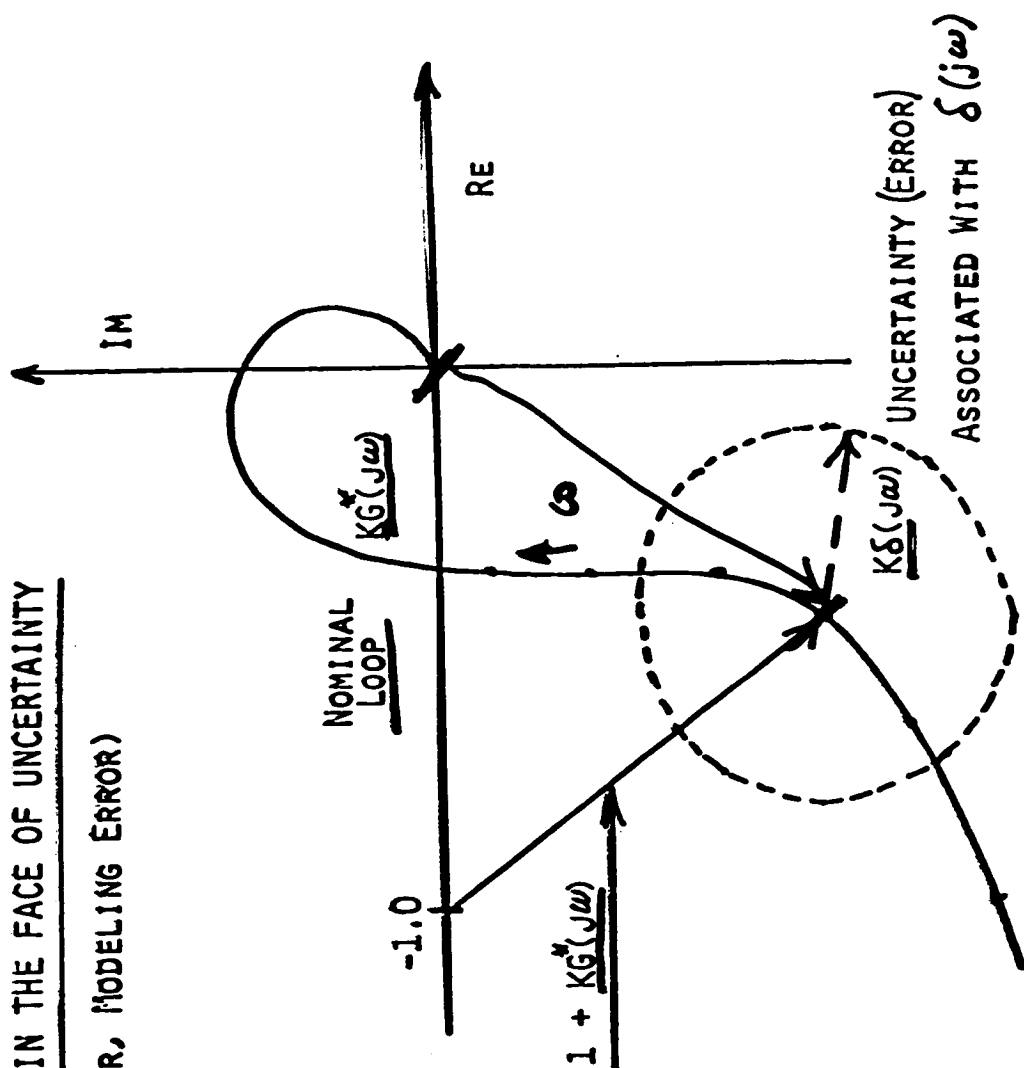
- Induce Critical Pole/Zero Interactions
- Significantly Affect Frequency Response Near Crossover
- Significantly Affect Time Response

CRITICAL DI-POLE EFFECT



STABILITY IN THE FACE OF UNCERTAINTY

(OR, MODELING ERROR)



→ DESIGN MODEL = $\underline{G^*(j\omega)}$

→ TRUE MODEL = $\underline{G^* + \delta}$

→ STABILITY GUARANTEED IF

$$|K\delta(j\omega)| < |1 + KG^*(j\omega)|$$

or if

$$|\epsilon| < |1 + (KG^*)|$$

with $\epsilon = (G^*)^{-1} \delta G$

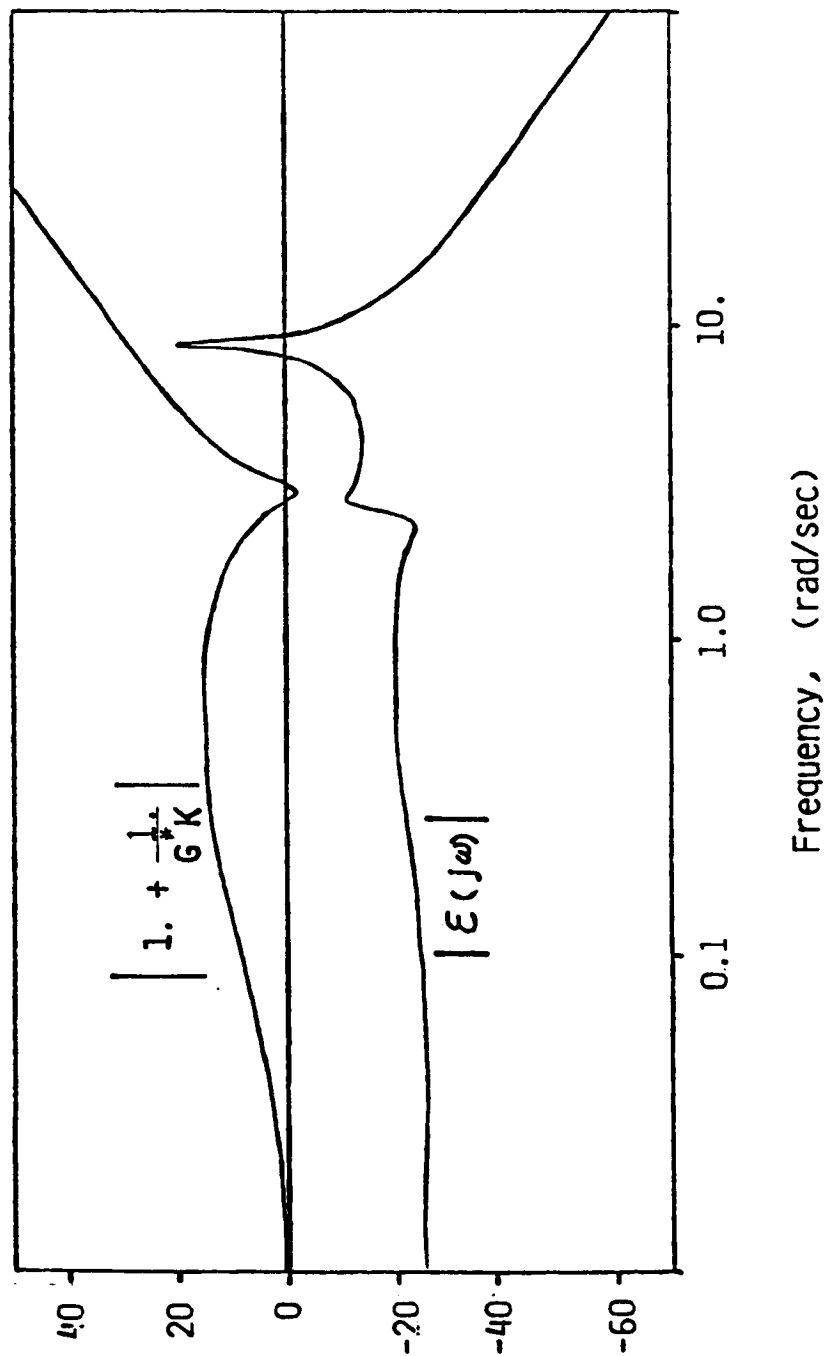
ROBUSTNESS EVALUATION

Slender-Body Configuration

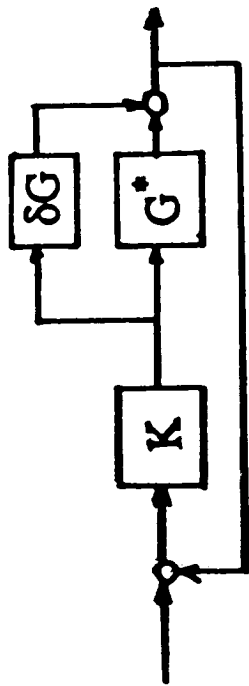
$G^* = \text{Rigid Model } (M=1.8, \text{ S.L.})$

$\omega_1 = 9 \text{ rad/sec}$

$\omega_{sp}^* = 7.2 \text{ rad/sec}$



CHARACTERIZING UNCERTAINTY

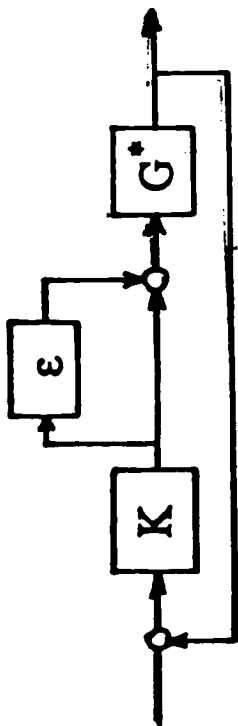


$$G = G^* + \delta G$$

— If N.U.P.(G) = N.U.P.(G*)

Stability of the Loop
Guaranteed If

$$\bar{\sigma}[\delta G K] < \underline{\sigma}[I + G K(j\omega)], \omega > 0$$



$$G = G^* (1 - \epsilon)$$

— If N.U.P.(G) = N.U.P.(G*)

Stability of the Loop
Guaranteed If

$$\bar{\sigma}[\epsilon] < \underline{\sigma}[I + [G K(j\omega)]^{-1}]$$

Most Critical / When $\|G K\| \approx 1$ (crossing)

→ Let's Look At Potential Sources For δG or $\epsilon(j\omega)$ ←

A MEANINGFUL METRIC FOR MODEL VALIDITY

$$\delta G = E(j\omega) = G(j\omega) - G_s(j\omega) \quad \text{In Crossover Region}$$

$$(\text{or } E(j\omega) = I - G^{-1}G_s)$$

Can Use Singular Values If Desired

$$\bar{\sigma}(E) = \bar{\lambda}^{1/2}(EE^*),$$

A Possible Metric

$$\|E(j\omega)\|_{CF} = \sup_{\omega_1 < \omega < \omega_2} \bar{\sigma}(E(j\omega))$$

A Conservative Metric

$$\|E(j\omega)\|_{\infty} = \sup_{0 < \omega < \infty} \bar{\sigma}(E(j\omega))$$

Or Just look at Bode Plots

CONSIDER THE SYSTEM IN POLYNOMIAL-MATRIX FORM

$$\begin{bmatrix} A(s) & c(s) \\ r(s) & m(s) \end{bmatrix} \begin{bmatrix} Z(s) \\ z_r(s) \end{bmatrix} = \begin{bmatrix} B(s) \\ b_r(s) \end{bmatrix} U(s)$$

$$Y(s) = M(s)Z(s) + m_r(s)z_r(s) \Rightarrow Y(s) = [G(s)]U(s)$$

$G_{ij}(s)$ Determined From

$$\frac{Z_i(s)}{U_j(s)} = \frac{m \det [A_i | B_j - cm^{-1}(r_i | b_{r_j})]}{m \det [A - cm^{-1}r]}$$

$$\frac{z_r(s)}{U_j(s)} = \frac{b_{r_j} \det [A - B_j b_{r_j}^{-1}r]}{m \det [A - cm^{-1}r]} \quad \left(\text{Assumes } z_r(s) \text{ Scalar} \right)$$

NOW ASSUME

$$c_k r_k \ll m \quad ; \quad k = 1, \dots, n$$

$$B_{jk} r_k \ll b_{r_j} \quad ; \quad k = 1, \dots, n$$

$$c_k (r_i | b_{r_j})_k \ll m \quad ; \quad k = 1, \dots, n$$

Over The Freq. Range Of Interest

In This Case

$$\left\{ \begin{array}{l} A_i | B_j - c m^{-1} (r_i | b_{r_j}) \approx A_i | B_j \\ A - B_j b_{r_j}^{-1} r \approx A \\ A - c m^{-1} r \approx A \end{array} \right.$$

And

$$\left\{ \begin{array}{l} \frac{Z_i(s)}{U_j(s)} \approx \frac{\hat{Z}_i(s)}{U_j(s)} = \frac{\det [A_i | B_j]}{\det [A]} \\ \frac{z_r(s)}{U_j(s)} \approx b_{r_j}/m \end{array} \right.$$

CASE I

If $\frac{b_{rj}}{m} \approx 0$ In Freq. of Interest

Truncated
Model

$$Y(s) \approx M(s)\hat{Z}(s)$$

$$\frac{\hat{Z}_i(s)}{U_j(s)} = \det[A_i | B_j] / \det[A]$$

Adequate Model When z_r
Corresponds To A Low-Frequency,
Or "Slow" Degree Of Freedom

Special Case - Modal Form

$$\begin{bmatrix} (sI - \Lambda) & 0 \\ 0 & (sI - \Lambda_r) \end{bmatrix} \begin{bmatrix} N(s) \\ N_r(s) \end{bmatrix} = \begin{bmatrix} B \\ B_r \end{bmatrix} U(s)$$

$$Y = M N(s) + M_r N_r(s)$$

Here

$$E(s) = G(s) - G_r(s) = M_r(sI - \Lambda_r)^{-1} B_r$$

$$E_{ij}(j\omega) \rightarrow 0 \text{ as } |(j\omega - \lambda_r)| \rightarrow \infty$$

CASE II

If $\frac{b_{rj}}{m} \rightarrow \text{Constant} (\neq 0)$ In Freq. of Interest

Residualized
Model

$$Y(s) \approx M(s)\hat{Z}(s) + m_r(s) \frac{b_r(0)}{k_r} U(s)$$

$$\frac{\hat{Z}_i(s)}{U_j(s)} = \frac{\det[A_i | B_j]}{\det[A]}$$

Adequate Model When Z_r Corresponds to
A (stable) High-Frequency ("Fast")
Degree of Freedom

Special Case

Modal $E(s) = M_r(sI - \Lambda_r)^{-1} B_r + M_r(\Lambda_r)^{-1} B_r$

Resid.

$$= M_r[s(I - \Lambda_r)^{-1} \Lambda_r^{-1}] B_r$$

$$|E_{ij}(j\omega)| \rightarrow 0 \text{ as } |j\omega| \rightarrow 0$$

Ch. 6. 1st
 B. 1st
 1st 1st

Table 6. Frequency Weighted Internally Balanced Reduction

Good AP
 1st 1st

Given: System state space description A, B, C and weighting filter state space description A_w, B_w, C_w .

Find: r^{th} order system

Step 1: Solve for X and Y

$$\begin{bmatrix} A & BC_w \\ 0 & A_w \end{bmatrix} \begin{bmatrix} X \\ X_{21} \end{bmatrix} + \begin{bmatrix} X \\ X_{21} \end{bmatrix} \begin{bmatrix} X_{12} & X_{22} \end{bmatrix} + \begin{bmatrix} A^T & 0 \\ C_w^T B^T A_w^T \end{bmatrix} \begin{bmatrix} 0 & 0 \\ 0 & B_w B_w^T \end{bmatrix} = 0$$

$$\begin{bmatrix} A^T & 0 \\ C_w^T B^T A_w^T \end{bmatrix} \begin{bmatrix} Y \\ Y_{21} \end{bmatrix} + \begin{bmatrix} Y \\ Y_{21} \end{bmatrix} \begin{bmatrix} Y_{12} & Y_{22} \end{bmatrix} + \begin{bmatrix} A & BC_w \\ 0 & A_w \end{bmatrix} \begin{bmatrix} CC^T & 0 \\ 0 & 0 \end{bmatrix} = 0$$



Step 2: Find T and Σ where $XY = T\Sigma^2T^{-1}$, $T = [T_r \ T_{n-r}]$, $T^{-T} = [U_r \ U_{n-r}]$

$$\Sigma^2 = \begin{bmatrix} \Sigma_r^2 & 0 \\ 0 & \Sigma_{n-r}^2 \end{bmatrix} \quad \text{where}$$

$$\Sigma_r = \text{diag}(v_{c_1} v_{o_1}) \quad i = 1, \dots, r$$

$$\Sigma_{n-r} = \text{diag}(v_{c_i} v_{o_i}) \quad i = r+1, \dots, n$$

$$v_{c_1} v_{o_1} \geq \dots \geq v_{c_n} v_{o_n} \geq 0$$

Step 3: r^{th} order system is

$$A_r = U_r^T A T_r$$

$$B_r = U_r^T B$$

$$C_r = C T_r$$

\bar{Y}_R (Rigid Body)
D.O.F.

THE SYSTEM IS DESCRIBED BY
(After Linearization)

$$\begin{bmatrix} [F_{RB}(s)] & 0 & 0 & \dots \\ [0] & s^2 + 2\xi_1\omega_1 s + \omega_1^2 & 0 & \dots \\ 0 & \vdots & \ddots & \vdots \end{bmatrix} \begin{bmatrix} \bar{Y}_R \\ \eta_1 \\ \vdots \\ \eta_m \end{bmatrix} = \begin{bmatrix} \bar{M}_R \\ F_1 \\ \vdots \\ F_m \end{bmatrix} \bar{\delta}_c(s)$$

[F(s)] [M(s)]

(Elastically Decoupled)

$$\left\{ \begin{array}{l} \text{Aero Coupling} \\ \text{(can include} \\ \text{unsteady effects)} \end{array} \right\} + \left\{ \begin{array}{l} [A_{RR}(s)] [A_{RE_1}(s)] \dots \\ [A_{E_1R}(s)] A_{E_1}(s) \dots \\ \vdots \vdots \vdots \end{array} \right\} \begin{bmatrix} \bar{Y}_R \\ \eta_1 \\ \vdots \\ \eta_m \end{bmatrix}$$

[A(s)]

a.g. $A_{E_i}(s) = \frac{N_{E_i}(s)}{D_{E_i}(s)}, (\text{Unsteady})$

FOR QUASI-STEADY AERODYNAMICS A STANDARD FORM IS

$$\underbrace{\begin{bmatrix} (s-Z_w) & -(Z_q+U_0) & \vdots \\ -M_w & (s-M_q) & \vdots \\ -F_{lw} & -F_{lq} & \vdots \end{bmatrix}}_{F_{RR}} \underbrace{\begin{bmatrix} -(Z_{\dot{\eta}_1} s + Z_{\eta_1}) & \dots \\ -(M_{\dot{\eta}_1} s + M_{\eta_1}) & \dots \\ \{s^2 + (2\xi_1 \omega_1 - F_{l1}) \frac{\eta_1}{\dot{\eta}_1}\} s + (\omega_1^2 - F_{l1} \eta_1) & \vdots \end{bmatrix}}_{F_{RE}} \underbrace{\begin{bmatrix} w \\ \dot{\theta} \\ \eta_1 \\ \vdots \end{bmatrix}}_{\delta} = \underbrace{\begin{bmatrix} Z_{\delta} \\ M_{\delta} \\ -F_{l\delta} \\ \vdots \end{bmatrix}}_{F_{\delta E}}$$

And, for example

$$\frac{\dot{\theta}(s)}{\delta_E(s)} = \frac{C_{\theta}(s+1/\tau_{\theta}) \prod_{i=1}^m (s^2 + 2\xi_{\theta i} \omega_{\theta i} s + \omega_{\theta i}^2)}{(s^2 + 2\xi_{sp} \omega_{sp} s + \omega_{sp}^2) \prod_{i=1}^m (s^2 + 2\xi_{Ei} \omega_{Ei} s + \omega_{Ei}^2)}$$

like Rigid body
but mod

Aerelastic
Dipoles

From
Cramer's
Rule

Table 4. Elastic Aircraft Linear Longitudinal Equations In Polynomial Matrix Form

$s - X_u$	$-X_\alpha$	$-X_q s - X_\theta$	$-X_{\eta_1} s - X_{\eta_2}$	$-X_{\eta_3} s - X_{\eta_4}$
$\frac{Z_u}{V_{T_1}}$	$s \frac{Z_\alpha}{V_{T_1}}$	$-(1 + \frac{Z_q}{V_{T_1}}) s - \frac{Z_\theta}{V_{T_1}}$	$\frac{Z_{\eta_1}}{V_{T_1}} s - \frac{Z_{\eta_2}}{V_{T_1}}$	$\frac{Z_{\eta_3}}{V_{T_1}} s - \frac{Z_{\eta_4}}{V_{T_1}}$
$-M_u$	$-M_\alpha$	$s^2 - M_q s$	$-M_{\eta_1} s - M_{\eta_2}$	$-M_{\eta_3} s - M_{\eta_4}$
$-F_{1_u}$	$-F_{1_\alpha}$	$-F_{1_q} s$	$\frac{s^2 + (2\zeta_1 \omega_1 - F_{1_{\eta_1}}) s + (\omega_1^2 - F_{1_{\eta_1}})}{s^2 + (2\zeta_1 \omega_1 - F_{1_{\eta_1}}) s + (\omega_1^2 - F_{1_{\eta_1}})}$	$-F_{1_{\eta_3}} s - F_{1_{\eta_4}}$
$-F_{2_u}$	$-F_{2_\alpha}$	$-F_{2_q} s$	$-F_{2_{\eta_1}} s - F_{2_{\eta_2}}$	$\frac{s^2 + (2\zeta_2 \omega_2 - F_{2_{\eta_3}}) s + (\omega_2^2 - F_{2_{\eta_3}})}{s^2 + (2\zeta_2 \omega_2 - F_{2_{\eta_3}}) s + (\omega_2^2 - F_{2_{\eta_3}})}$
$-K_u$	$-K_\alpha$	$-K_q s$	$-K_{\eta_1} s - K_{\eta_2}$	$-K_{\eta_3} s - K_{\eta_4}$
0	0	$-s$	$\phi_1 s$	$\phi_2 s$

where

$$K_u = Z_u - x_p M_u + \sum_{i=1}^n \phi_i F_{1_i}$$

$$K_\alpha = Z_\alpha - x_p M_\alpha + \sum_{i=1}^n \phi_i F_{1_i}$$

$$K_q = Z_q - x_p M_q + \sum_{i=1}^n \phi_i F_{1_i}$$

$$K_{\eta_1} = Z_{\eta_1} - x_p M_{\eta_1} - \phi_1 (\omega_1^2 - F_{1_{\eta_1}}) + \sum_{j=1}^n \phi_j F_{1_{\eta_j}}$$

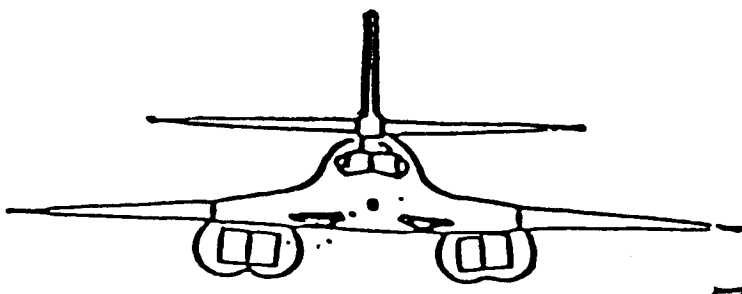
$$K_{\eta_2} = Z_{\eta_2} - x_p M_{\eta_2} - \phi_1 (2\zeta_1 \omega_1 - F_{1_{\eta_1}}) + \sum_{j=1}^n \phi_j F_{1_{\eta_j}}$$

$$K_{\eta_3} = Z_{\eta_3} - x_p M_{\eta_3} + \sum_{i=1}^n \phi_i F_{1_{\eta_i}}$$

$$K_{\eta_4} = Z_{\eta_4} - x_p M_{\eta_4} + \sum_{i=1}^n \phi_i F_{1_{\eta_i}}$$

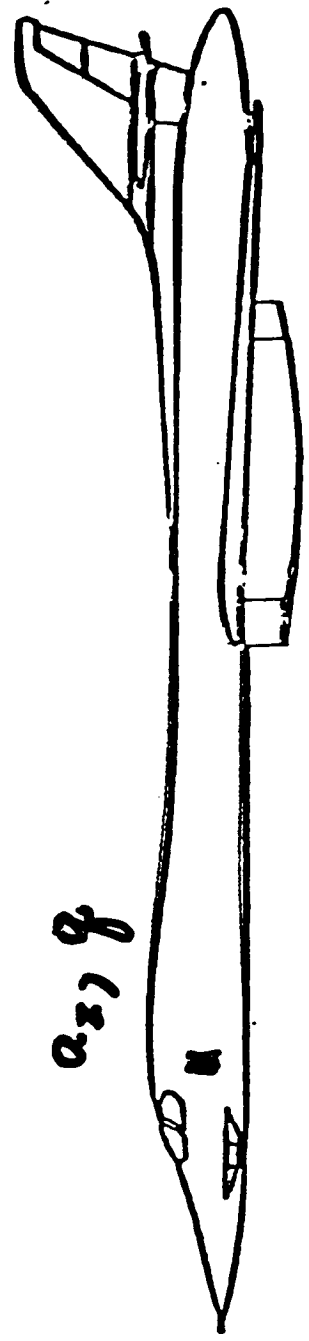
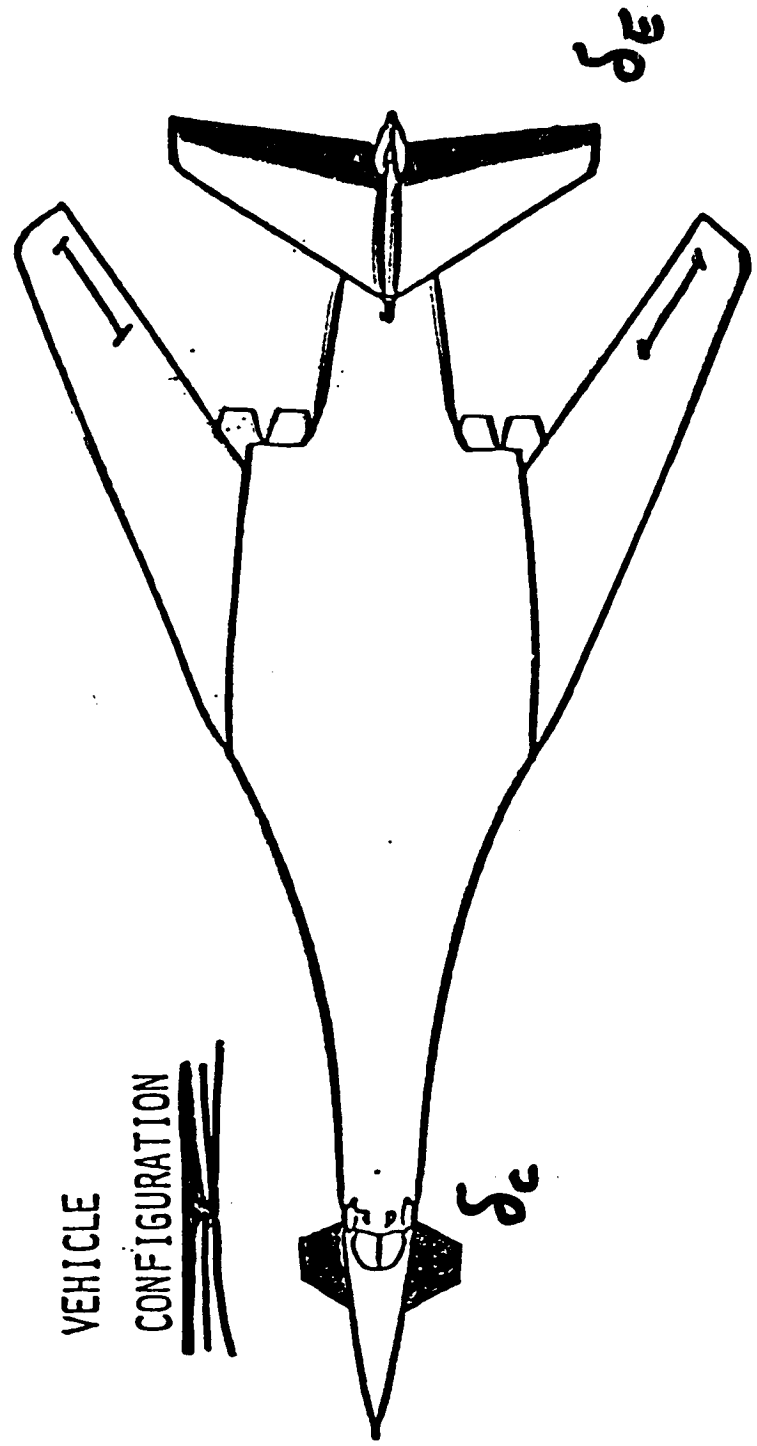
$$u = \begin{Bmatrix} \delta \epsilon \\ \delta c \end{Bmatrix}$$

$$Y = [G(s)] U(s)$$



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VEHICLE
CONFIGURATION



$\omega s/2$ 1-9

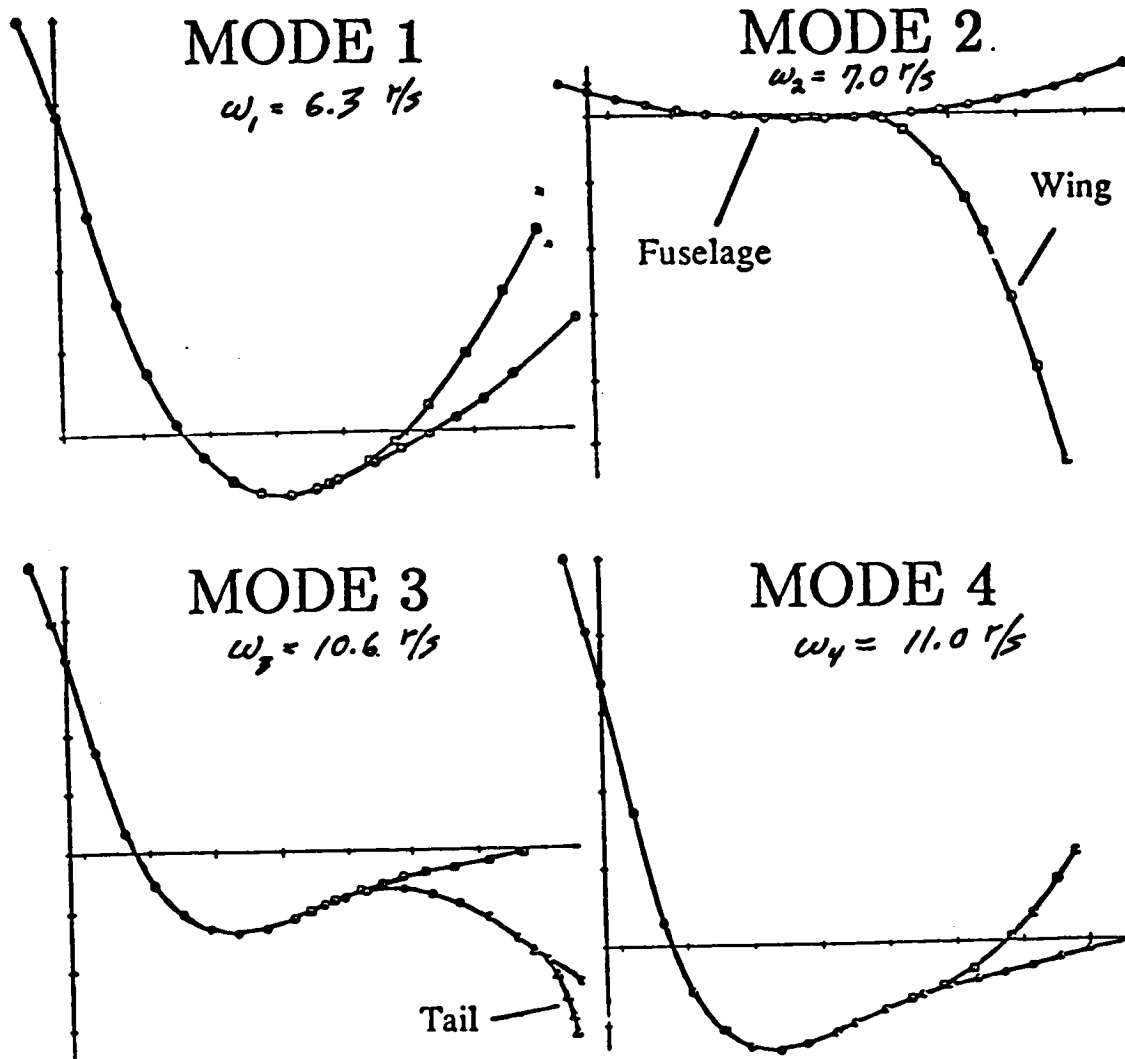
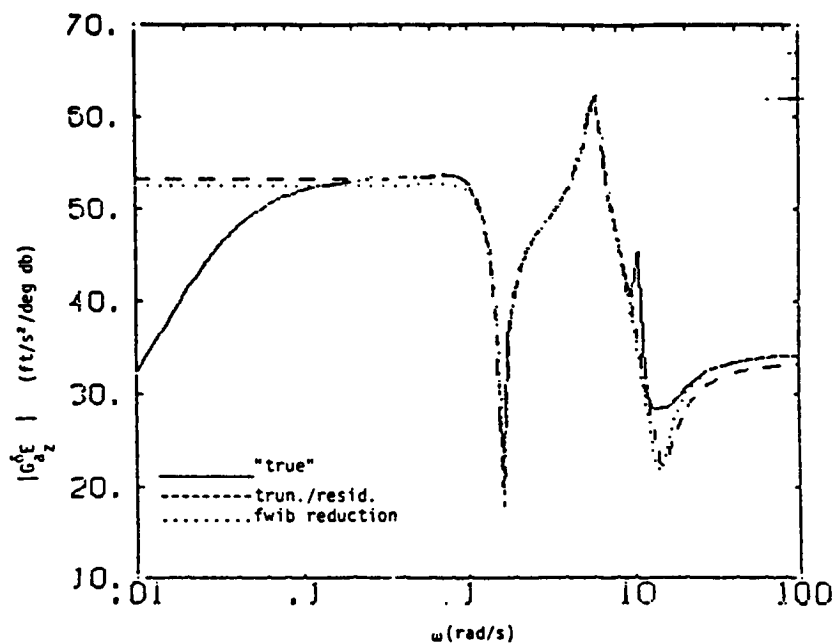


Table 10. Transfer Functions For the True Model

	$G_{a_z}^{\delta_B}$ (ft/s ² /deg)	$G_{q_p}^{\delta_B}$ (rad/s/deg)	$G_{a_z}^{\delta_c}$ (ft/s ² /deg)	$G_{q_p}^{\delta_c}$ (rad/s/deg)
gains	52.01	8.001	-244.5	15.65
zeros	6.473E-5 -.008887 -.01958 ± j1.661 -.3610 ± j11.00 -1.003 ± j11.13 1.549 ± j11.71 -3.144 ± j14.34	0 -.05103 -.2020 3.642 -4.020 -.3610 ± j11.00 -2.838 ± j12.71 .5735 ± j13.41	1.087E-4 -.008093 .1703 ± j1.795 -.8996 ± j4.132 -.2252 ± j10.77 -.3607 ± j11.00 -2.601 ± j13.06	0 -.05541 -.1172 -.5973 ± j2.912 -.2556 ± j10.68 -.3562 ± j10.99 -2.564 ± j13.12
poles	.03324 -.04268 -.4513 ± j1.171 -.4408 ± j6.010 -.2240 ± j10.78 -.3611 ± j11.00 -2.558 ± j13.05			

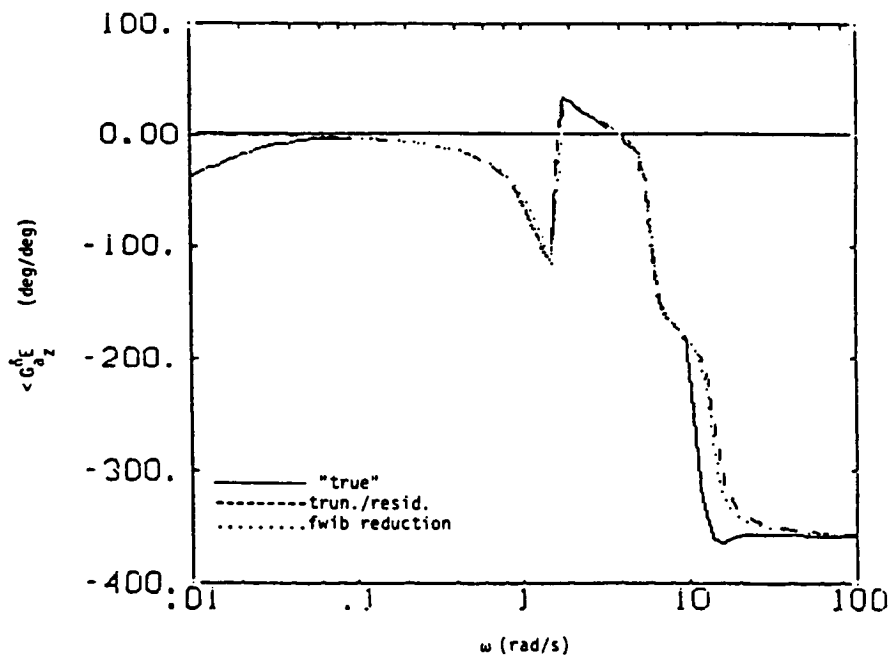
Bode Magnitude

$$G_{a_z}^{\delta E}$$



Bode Phase

$$G_{a_z}^{\delta E}$$

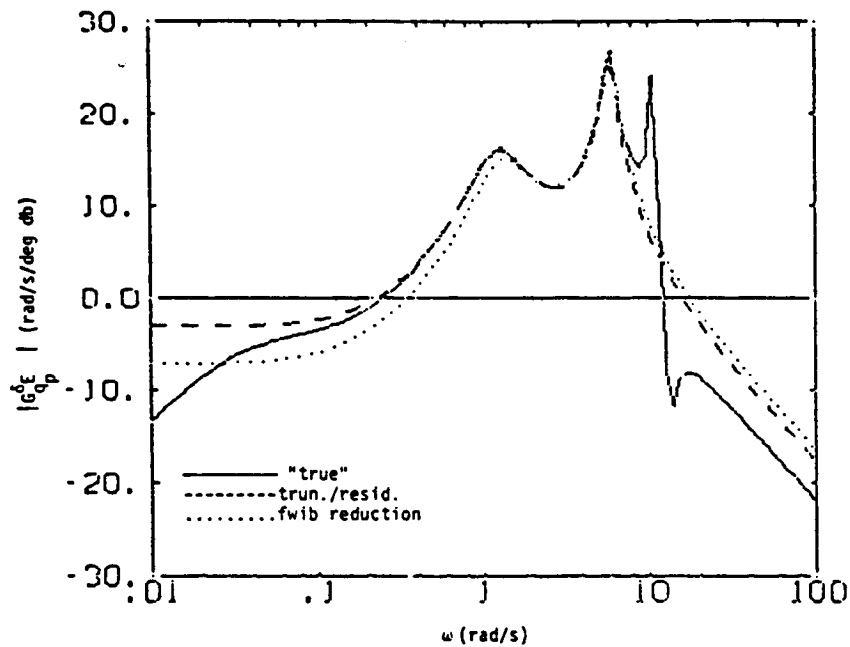


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Figure 4. $\angle G_{a_z}^{\delta E}$ Frequency Response

Bode Magnitude

$$G_{\delta E}^{\delta E}$$



Bode Phase

$$G_{\delta E}^{\delta E}$$

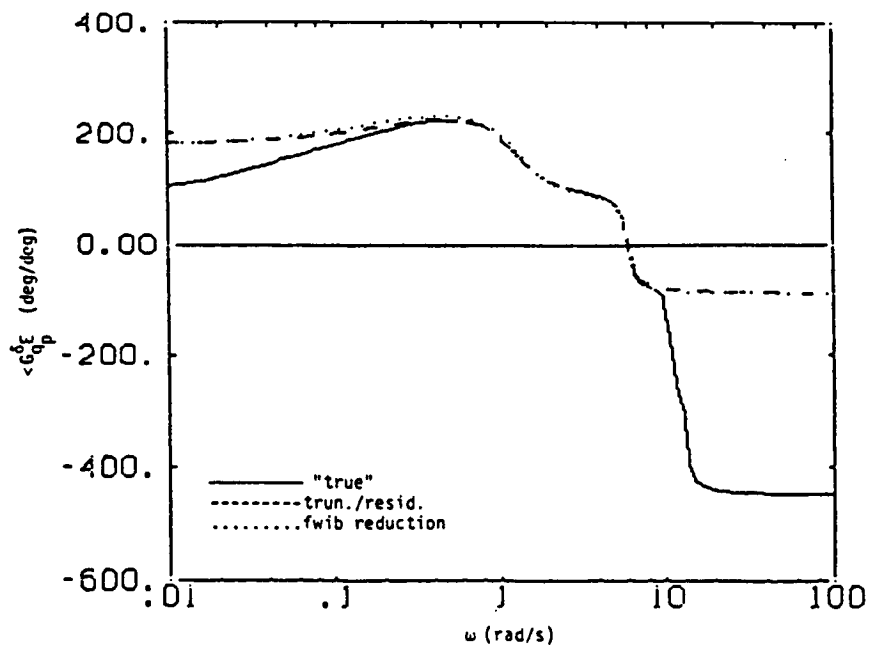
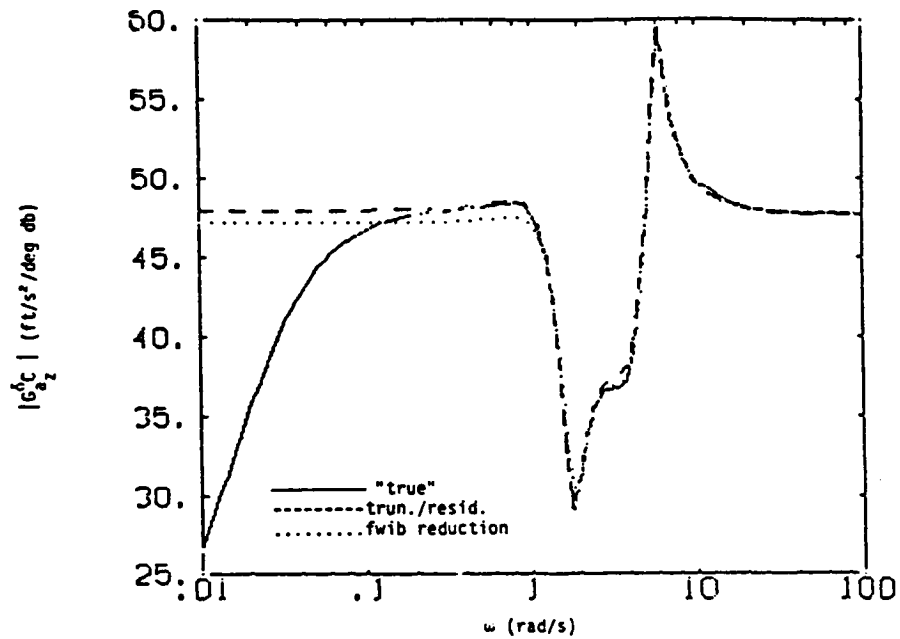


Figure 6. $\angle G_{\delta E}^{\delta E}$ Frequency Response

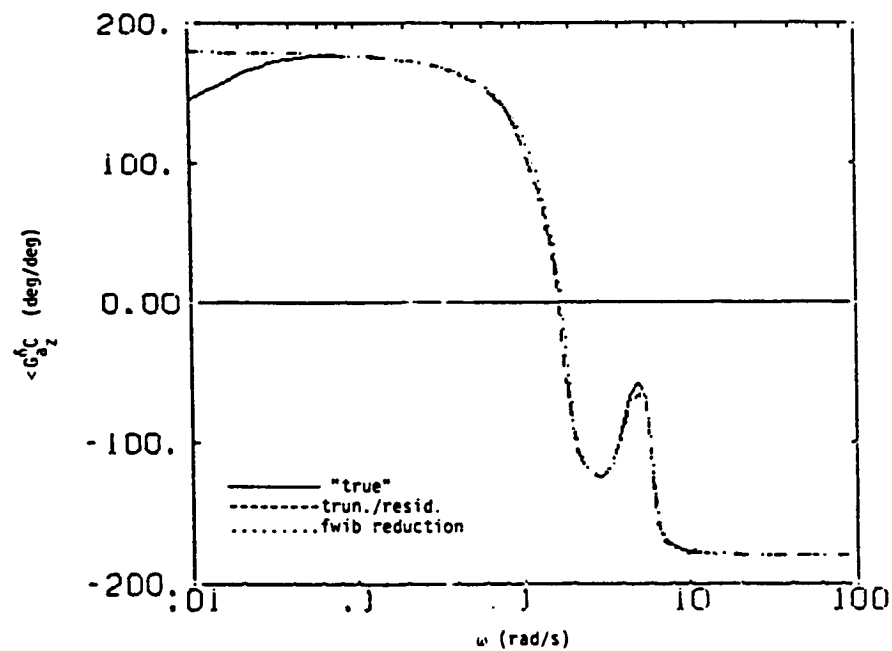
Bode Magnitude

$G_{a_z}^{\delta_c}$

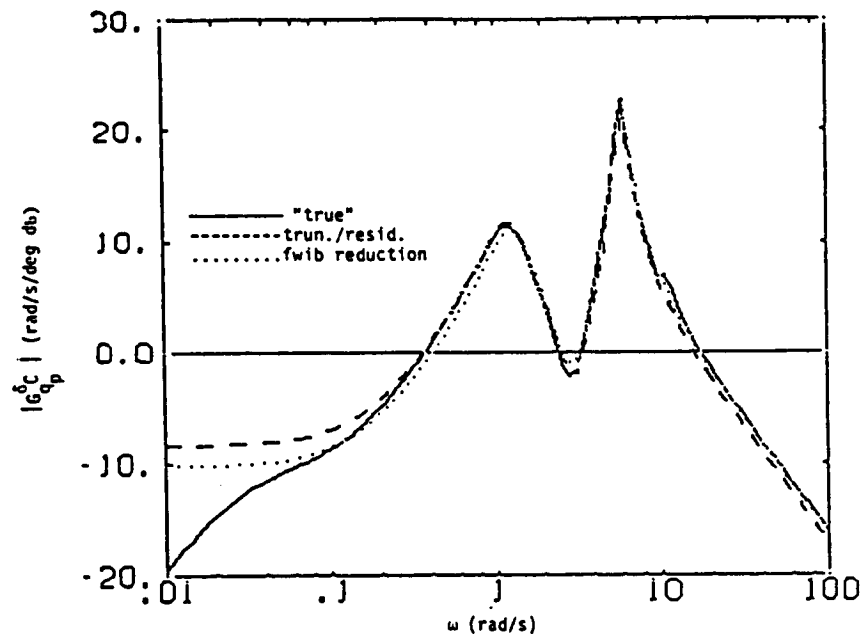


Bode Phase

$G_{a_z}^{\delta_c}$



Bode Magnitude $G_{\delta}^{\delta_c}$



Bode Phase $G_{\delta}^{\delta_c}$

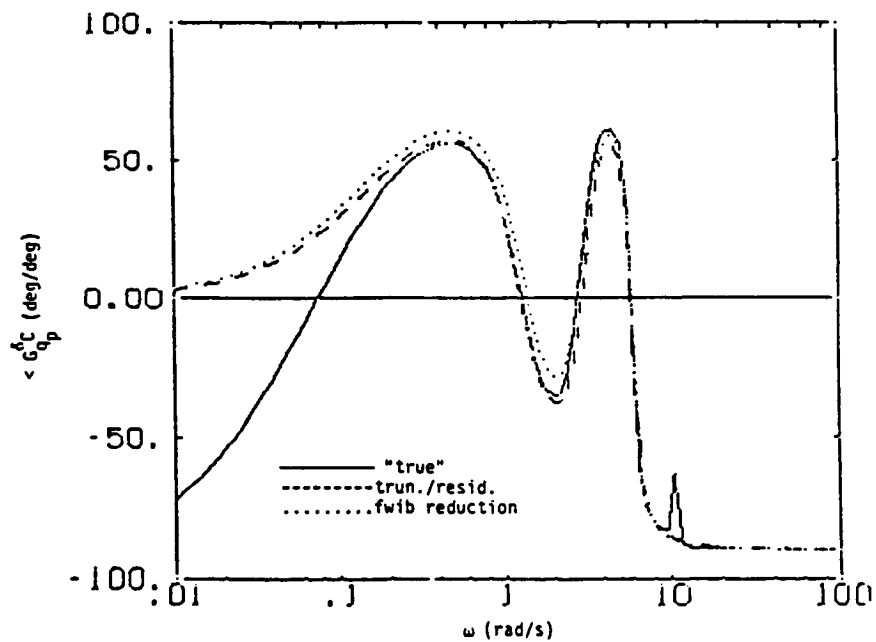


Figure 10. $\angle G_{\delta}^{\delta_c}$ Frequency Response

Table 12. Transfer Functions For FWIB Reduction Model

	$G_{a_z}^{\delta a}$ (ft/s ² /deg)	$G_{q_p}^{\delta a}$ (rad/s/deg)	$G_{a_z}^{\delta \epsilon}$ (ft/s ² /deg)	$G_{q_p}^{\delta \epsilon}$ (rad/s/deg)
gains	52.01	14.96	-244.5	15.29
zeros	-.02102 ± j1.670 1.244 ± j13.51	-.1789 2.781 -3.732	.1725 ± j1.806 -.9177 ± j4.143	-.1437 -.6806 ± j2.900
poles	-.4679 ± j1.234 -.4413 ± j6.015			

Table 14. Lower-Order Transfer Function Forms

$$G_{a_z}^{\delta_E}(s) = \frac{K_{a_z}^{\delta_E} s[s^2 + s_p(2\zeta\omega)_{a_z}^{\delta_E} s + s_p(\omega^2)_{a_z}^{\delta_E}][s^2 + f_1(2\zeta\omega)_{a_z}^{\delta_E} s + f_1(\omega^2)_{a_z}^{\delta_E}]}{D(s)}$$

$$G_{q_p}^{\delta_E}(s) = \frac{K_{q_p}^{\delta_E} s[s + s_p(\frac{1}{T})_{q_p}^{\delta_E}][s + f_{11}(\frac{1}{T})_{q_p}^{\delta_E}][s + f_{12}(\frac{1}{T})_{q_p}^{\delta_E}]}{D(s)}$$

$$G_{a_z}^{\delta_c}(s) = \frac{K_{a_z}^{\delta_c} s[s^2 + s_p(2\zeta\omega)_{a_z}^{\delta_c} s + s_p(\omega^2)_{a_z}^{\delta_c}][s^2 + f_1(2\zeta\omega)_{a_z}^{\delta_c} s + f_1(\omega^2)_{a_z}^{\delta_c}]}{D(s)}$$

$$G_{q_p}^{\delta_c}(s) = \frac{K_{q_p}^{\delta_c} s[s + s_p(\frac{1}{T})_{q_p}^{\delta_c}][s^2 + f_1(2\zeta\omega)_{q_p}^{\delta_c} s + f_1(\omega^2)_{q_p}^{\delta_c}]}{D(s)}$$

where $D(s) = s[s^2 + (2\zeta\omega)_{sp}s + (\omega^2)_{sp}][s^2 + (2\zeta\omega)_{f1}s + (\omega^2)_{f1}]$

Table 7. Truncated/Residualized Model

$\left[s - \frac{Z_\alpha}{V_{T_1}} \right]$	$-(1 + \frac{Z_q}{V_{T_1}})s - \frac{Z_\theta}{V_{T_1}}$	$-\frac{Z_{\eta_1}}{V_{T_1}}s - \frac{Z_{\eta_1}}{V_{T_1}}$	0	0	$\alpha(s)$	$\frac{Z_{\delta_B}}{V_{T_1}}$	$\frac{Z_{\delta_C}}{V_{T_1}}$	$\delta_E(s)$
$-M_\alpha$	$s^2 - M_q s$	$-M_{\eta_1} s - M_{\eta_1}$	0	0	$\theta(s)$	M_{δ_B}	M_{δ_C}	$\delta_C(s)$
$-F_{1_\alpha}$	$-F_{1_q} s$	$s^2 + (2\zeta_1 \omega_1 - F_{1_{\eta_1}})s + (\omega_1^2 - F_{1_{\eta_1}})$	0	0	$\eta_1(s)$	$F_{1_{\delta_B}}$	$F_{1_{\delta_C}}$	
$-K_\alpha$	$-K_q s$	$-K_{\eta_1} s - K_{\eta_1}$	1	0	$a_z(s)$	K_{δ_B}	K_{δ_C}	
0	-s	$\phi_i s$	0	1	$q_p(s)$	0	0	

=

$$\begin{aligned}
D(s) = & -\left(\frac{Z_{\eta_1}}{V_{T_1}}s + \frac{Z_{\eta_1}}{V_{T_1}}\right)[M_\alpha F_{1_q}s + F_{1_\alpha}(s^2 - M_q s)] \\
& - (M_{\eta_1}s + M_{\eta_1})\left[F_{1_q}s - \frac{Z_\alpha}{V_{T_1}}\right] + \left(1 + \frac{Z_q}{V_{T_1}}\right)F_{1_\alpha}s \\
& + (s^2 + (2\zeta_1\omega_1 - F_{1_{\eta_1}})s + (\omega_1^2 - F_{1_{\eta_1}}))\left[\left(s - \frac{Z_\alpha}{V_{T_1}}\right)(s^2 - M_q s) - \left(1 + \frac{Z_q}{V_{T_1}}\right)M_\alpha s\right]
\end{aligned} \tag{23}$$

$$\begin{aligned}
N(s) = & \frac{Z_{\delta_B}}{V_{T_1}}\{s[M_\alpha(s^2 + (2\zeta_1\omega_1 - F_{1_{\eta_1}})s + (\omega_1^2 - F_{1_{\eta_1}})) + F_{1_\alpha}(M_{\eta_1}s + M_{\eta_1})] \\
& - \phi_1 s[M_\alpha F_{1_q}s + F_{1_\alpha}(s^2 - M_q s)]\} \\
& + M_{\delta_B}\left\{s\left[\left(s - \frac{Z_\alpha}{V_{T_1}}\right)(s^2 + (2\zeta_1\omega_1 - F_{1_{\eta_1}})s + (\omega_1^2 - F_{1_{\eta_1}})) - F_{1_\alpha}\left(\frac{Z_{\eta_1}}{V_{T_1}}s + \frac{Z_{\eta_1}}{V_{T_1}}\right)\right]\right. \\
& \left. - \phi_1 s\left[F_{1_q}s - \frac{Z_\alpha}{V_{T_1}}\right] + \left(1 + \frac{Z_q}{V_{T_1}}\right)F_{1_\alpha}s\right\} \\
& + F_{1_{\delta_B}}\left\{s\left[\left(s - \frac{Z_\alpha}{V_{T_1}}\right)(M_{\eta_1}s + M_{\eta_1}) + M_\alpha\left(\frac{Z_{\eta_1}}{V_{T_1}}s + \frac{Z_{\eta_1}}{V_{T_1}}\right)\right]\right. \\
& \left. - \phi_1 s\left[\left(s - \frac{Z_\alpha}{V_{T_1}}\right)(s^2 - M_q s) - M_\alpha\left(1 + \frac{Z_q}{V_{T_1}}\right)s\right]\right\}
\end{aligned}$$

Table 8. Polynomial Coefficient and Factor Relationships

$$\begin{aligned}
 \tilde{d}_1 &= (\tilde{\omega}^2)_{sp} (\tilde{\omega}^2)_{f1} \\
 \tilde{d}_2 &= (2\tilde{\zeta}\omega)_{sp} (\tilde{\omega}^2)_{f1} + (2\tilde{\zeta}\omega)_{f1} (\tilde{\omega}^2)_{sp} \\
 \tilde{d}_3 &= (\tilde{\omega}^2)_{sp} + (\tilde{\omega}^2)_{f1} + (2\tilde{\zeta}\omega)_{sp} (2\tilde{\zeta}\omega)_{f1} \\
 \tilde{d}_4 &= (2\tilde{\zeta}\omega)_{sp} + (2\tilde{\zeta}\omega)_{f1} \\
 \tilde{n}_1 &= \tilde{K}_{q_p}^{\delta_B} \left[\left(\frac{1}{T} \right)_{q_p} f_{11} \left(\frac{1}{T} \right)_{q_p} f_{12} \left(\frac{1}{T} \right)_{q_p} \right] \\
 \tilde{n}_2 &= \tilde{K}_{q_p}^{\delta_B} \left[\left(\frac{1}{T} \right)_{q_p} f_{11} \left(\frac{1}{T} \right)_{q_p} + sp \left(\frac{1}{T} \right)_{q_p} f_{12} \left(\frac{1}{T} \right)_{q_p} + f_{11} \left(\frac{1}{T} \right)_{q_p} f_{12} \left(\frac{1}{T} \right)_{q_p} \right] \\
 \tilde{n}_3 &= \tilde{K}_{q_p}^{\delta_B} \left[\left(\frac{1}{T} \right)_{q_p} + f_{11} \left(\frac{1}{T} \right)_{q_p} + f_{12} \left(\frac{1}{T} \right)_{q_p} \right] \\
 \tilde{n}_4 &= \tilde{K}_{q_p}^{\delta_B}
 \end{aligned}$$

Table 9. Polynomial Coefficient and Factor Difference Relationships

$$\Delta \tilde{d}_1 = \tilde{\omega}_{sp}^2 \Delta(\tilde{\omega}^2)_{f1} + (\tilde{\omega}^2)_{f1} \Delta(\tilde{\omega}^2)_{sp}$$

$$\Delta \tilde{d}_2 = (2\tilde{\zeta}\omega)_{sp} \Delta(\tilde{\omega}^2)_{f1} + (\tilde{\omega}^2)_{f1} \Delta(2\tilde{\zeta}\omega)_{sp} + (2\tilde{\zeta}\omega)_{f1} \Delta(\tilde{\omega}^2)_{sp} + (\tilde{\omega}^2)_{sp} \Delta(2\tilde{\zeta}\omega)_{f1}$$

$$\Delta \tilde{d}_3 = \Delta(\tilde{\omega}^2)_{sp} + \Delta(\tilde{\omega}^2)_{f1} + (2\tilde{\zeta}\omega)_{sp} \Delta(2\tilde{\zeta}\omega)_{f1} + (2\tilde{\zeta}\omega)_{f1} \Delta(2\tilde{\zeta}\omega)_{sp}$$

$$\Delta \tilde{d}_4 = \Delta(2\tilde{\zeta}\omega)_{sp} + \Delta(2\tilde{\zeta}\omega)_{f1}$$

$$\begin{aligned} \Delta \tilde{n}_1 = & \tilde{K}_{q_p} \left(\frac{1}{T} \right)^{\delta_B} sp \left(\frac{1}{T} \right)^{\delta_B} f_{11} \left(\frac{1}{T} \right)^{\delta_B} \Delta \left(\frac{1}{T} \right)^{\delta_B} f_{12} \left(\frac{1}{T} \right)^{\delta_B} sp \left(\frac{1}{T} \right)^{\delta_B} f_{12} \left(\frac{1}{T} \right)^{\delta_B} \Delta \left(\frac{1}{T} \right)^{\delta_B} f_{11} \left(\frac{1}{T} \right)^{\delta_B} q_p \\ & + \tilde{K}_{q_p} f_{11} \left(\frac{1}{T} \right)^{\delta_B} sp \left(\frac{1}{T} \right)^{\delta_B} f_{12} \left(\frac{1}{T} \right)^{\delta_B} \Delta \left(\frac{1}{T} \right)^{\delta_B} sp \left(\frac{1}{T} \right)^{\delta_B} f_{11} \left(\frac{1}{T} \right)^{\delta_B} q_p f_{12} \left(\frac{1}{T} \right)^{\delta_B} \Delta \tilde{K}_{q_p} \delta_B \end{aligned}$$

$$\begin{aligned} \Delta \tilde{n}_2 = & \tilde{K}_{q_p} \left[sp \left(\frac{1}{T} \right)^{\delta_B} + f_{11} \left(\frac{1}{T} \right)^{\delta_B} \right] \Delta \left(\frac{1}{T} \right)^{\delta_B} f_{12} \left(\frac{1}{T} \right)^{\delta_B} \Delta \left(\frac{1}{T} \right)^{\delta_B} sp \left(\frac{1}{T} \right)^{\delta_B} f_{12} \left(\frac{1}{T} \right)^{\delta_B} \Delta \left(\frac{1}{T} \right)^{\delta_B} q_p \\ & + \tilde{K}_{q_p} \left[f_{11} \left(\frac{1}{T} \right)^{\delta_B} + f_{12} \left(\frac{1}{T} \right)^{\delta_B} \right] \Delta \left(\frac{1}{T} \right)^{\delta_B} sp \left(\frac{1}{T} \right)^{\delta_B} f_{11} \left(\frac{1}{T} \right)^{\delta_B} q_p \\ & + sp \left(\frac{1}{T} \right)^{\delta_B} f_{12} \left(\frac{1}{T} \right)^{\delta_B} q_p + f_{11} \left(\frac{1}{T} \right)^{\delta_B} f_{12} \left(\frac{1}{T} \right)^{\delta_B} \Delta \tilde{K}_{q_p} \delta_B \end{aligned}$$

$$\Delta \tilde{n}_3 = \tilde{K}_{q_p} \Delta f_{12} \left(\frac{1}{T} \right)^{\delta_B} + \tilde{K}_{q_p} \Delta f_{11} \left(\frac{1}{T} \right)^{\delta_B} + \tilde{K}_{q_p} \Delta sp \left(\frac{1}{T} \right)^{\delta_B} + [sp \left(\frac{1}{T} \right)^{\delta_B} + f_{11} \left(\frac{1}{T} \right)^{\delta_B} + f_{12} \left(\frac{1}{T} \right)^{\delta_B}] \Delta \tilde{K}_{q_p} \delta_B$$

$$\Delta \tilde{n}_4 = \Delta \tilde{K}_{q_p} \delta_B$$

Table 14. Transfer Functions from Literal Approximations

	$G_{a_z}^{\delta a}$ (ft/s ² /deg)	$G_{q_p}^{\delta a}$ (rad/s/deg)	$G_{a_z}^{\delta e}$ (ft/s ² /deg)	$G_{q_p}^{\delta e}$ (rad/s/deg)
gains	46.43	13.06	-241.5	14.06
zeros	0 -.1301±j2.039 1.854±j14.25	0 -.2038 3.497 -3.946	0 .1092±j1.834 -.7248±j4.006	0 -.1592 -.7430±j2.825
poles	0 -.4497±j1.246 -.4838±j6.050			

CLOSED-FORM EXPRESSIONS FOR THESE TERMS

Specifically, it has been shown that for a slender (low aspect ratio) vehicle

$$\omega_{sp}^{*2} - \omega_{sp}^2 = \frac{(U_o + Z_q)M_{\eta_1} F_{1_w}}{(\omega_1^2 - F_{1_{\eta_1}}) + (U_o + Z_q)M_w}$$

$$\omega_{I_E}^2 - \omega_{\theta_1}^2 = \frac{(U_o + Z_q)M_{\eta_q} F_{1_w}}{(\omega_1^2 - F_{1_{\eta_s}}) + (U_o + Z_q)M_w} - M_{\eta_1} F_{1\delta}/M_\delta$$

$$2\xi_{E_1} \omega_{E_1} = (2\xi_1 \omega_1 - F_{1_{\eta_1}}) + \frac{Z_{\eta_1} F_{1_w} + M_{\eta_1} F_{1_q}}{(\omega_1^2 - F_{1_{\eta_1}}) + (U_o + Z_q)M_w}$$

FURTHERMORE, IT CAN BE SHOWN

For Example, That

$$M_{\eta_i} = \frac{1}{2} \rho V^2 S \bar{c} C_{m_{\eta_i}}$$

$$C_{m_{\eta_i}} = \frac{1}{S \bar{c}} \left\{ \int_{-b/2}^{b/2} c_{l_{\alpha_w}} \left(\frac{\Delta x_w}{c_w} \right) \left(\frac{d\phi_i}{dx} \right)_w c_w^2 dy_w + C_{L_{\alpha_e}} \left(\frac{\Delta x_c}{\bar{c}} \right) \left(\frac{d\phi_i}{dx} \right)_c S \bar{c} \right\}$$

where

$c_{l_{\alpha_w}}$ = wing section lift-curve slope

$\left(\frac{d\phi_i}{dx} \right)_{(\cdot)}$ = mode i slope at location (\cdot)

SUMMARY

- Key Issues In Feedback Systems Reviewed
- Physics Of Model Uncertainty Addressed
- Mode Frequencies Near Crossover
- Mode Shapes And Dipoles
- With Aeroelastic Coupling, Both Affected
- Closed-Form Expressions Developed To Show Sources Of Interactions
- Sensitivity To Uncertainty In These Parameter May Be Further Explored

} Purely Elastic